

Rotary motors and helices.

Helice $\vec{z} = (R \cos(\lambda \ell), R \sin(\lambda \ell), \frac{P}{2\pi} \lambda \ell)$

$$\left| \frac{d\vec{z}}{d\ell} \right| = 1 \quad \lambda = \frac{1}{\sqrt{R^2 + \left(\frac{P}{2\pi}\right)^2}}$$

R - radius, $\frac{P}{2\pi}$ - pitch.

$$\vec{t} = \frac{d\vec{z}}{d\ell} = \left(-R\lambda \sin \lambda \ell, R\lambda \cos \lambda \ell, \frac{P\lambda}{2\pi} \right)$$

$$\frac{d\vec{t}}{d\ell} = \left(-R\lambda^2 \cos \lambda \ell, -R\lambda^2 \sin \lambda \ell, 0 \right)$$

$$K = R\lambda^2; \quad \vec{n} = (\cos \lambda \ell, \sin \lambda \ell, 0)$$

$$\frac{d\vec{n}}{d\ell} = K \vec{t} + \vec{\tau} \vec{b} \quad (\vec{b} - \text{binormal})$$

$$\vec{b} = \left(-\frac{P\lambda}{2\pi} \sin(\lambda \ell), \frac{P\lambda}{2\pi} \cos(\lambda \ell), -R\lambda \right)$$

$$\vec{\tau} = \frac{P\lambda^2}{2\pi}$$

Helice rotate with angular velocity (σ, σ, ω)

Resistive force coefficient approximation

$$F_z = \int_0^L d\ell (-S_{||} t_z v_t - S_{\perp} b_z v_b)$$

$$v_t = R^2 \lambda \omega$$

$$v_b = \frac{PR\lambda}{2\pi} \omega$$

Force on the rotating helix

$$F_z = L (S_{\perp} - S_{\parallel}) \frac{\omega P R^2 \lambda^2}{2\pi}$$

Helix angle

$$\cos \psi = \frac{P}{2\pi \lambda}$$

Projection length l ; $L = \frac{l}{\cos \psi}$

$$F_z = l (S_{\perp} - S_{\parallel}) \frac{P}{2\pi} \frac{(1 - \cos^2 \psi)}{\cos \psi} \omega$$

Torque

$$\vec{M}_z = - \int_0^L dl [\vec{e} \times (S_{\parallel} v_t \vec{t} + S_{\perp} v_b \vec{b})]$$

$$M_z = - \frac{l \omega}{\cos \psi} \left(\frac{P}{2\pi} \right)^2 (1 - \cos^2 \psi) (S_{\parallel} \tan^2 \psi + S_{\perp})$$

$\neq f$ helix moves with velocity $\vec{v} = v \vec{e}_z$ Here is torque

on helix

$$\vec{M}_z = - \int_0^L dl \vec{e} \times (S_{\parallel} t_z^2 + S_{\perp} b_z^2)$$

$$M_z = - \int_0^L dl (S_{\parallel} t_z [\vec{e} \times \vec{t}]_z + S_{\perp} b_z [\vec{e} \times \vec{b}]_z) v$$

$$= l \frac{P}{2\pi} \frac{(1 - \cos^2 \psi)}{\cos \psi} (S_{\perp} - S_{\parallel}) v$$

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Finally friction due to motion of the helix

$$F_z = - \int_0^L dt (S_{11} t_z^2 + S_{\perp} b_z^2) V =$$

$$= - V \frac{e}{\cos \psi} (S_{11} \cos^2 \psi + S_{\perp} (1 - \cos^2 \psi))$$

F_z, M_z are force and torque on the body from the liquid

$$\begin{pmatrix} F_z \\ M_z \end{pmatrix} = \begin{pmatrix} -A & B \\ C & D \end{pmatrix} \begin{pmatrix} v \\ \omega \end{pmatrix}$$

$\begin{pmatrix} A & B \\ C & D \end{pmatrix}$ - propulsion matrix according to Purcell, PNAS, 94, 11307 (1997)

Purcell proves $BC \geq 0 \Rightarrow B=C$
 For our helix

$$B=C = e \frac{p}{2\pi} \frac{(1 - \cos^2 \psi)}{\cos \psi} (S_{\perp} - S_{11}) V$$

Efficiency of propulsion according to Purcell



$$Bv - D\omega + T_m = 0$$

$$-A_0v + F = 0$$

$$-Av + B\omega - F = 0$$

$$-D_0\Omega - T_m = 0$$

$$\Omega_m = \omega - \Omega$$

Efficiency

$$E = \frac{A_0 v^2}{T_m \Omega_m} = \frac{A_0 D_0 B^2}{[D(A_0 + A) - B^2][A + A_0(D + D_0) - B^2]}$$

$$B^2 \ll AD$$

$$\frac{D_0}{D + D_0} \sim 1$$

$$E = \frac{A_0 B^2}{(A_0 + A)^2 D}$$

$$E_{max} = \frac{B^2}{4AD} \quad \text{- depends only on the shape of propeller.}$$

Left-handed helix rotating counter-clockwise looking along z axis.
Magnetotactic bacteria ~~are~~ have two directions of motion by switching between connected flagella with switch of rotating flagella.

Spinning flagella around bacterium

(Fig. 1)

(Fig. 2)

(Fig. 3)

Stokes flow - Stokes solution for sphere moving with \hat{u}

$$-\nabla p + \eta \Delta \vec{u} + \frac{1}{2} \zeta \text{rot} \vec{u} = 0; \quad \text{div} \vec{u} = 0$$

$$-\nabla \hat{p} + \eta \Delta \hat{u} = 0; \quad \text{div} \hat{u} = 0$$

Reciprocal theorem

$$\int \hat{u}_i \partial_{ik} h_k dS = \int h_i \partial_{ik} \hat{u}_k dS + \int \vec{z} \cdot \hat{\vec{z}}_0 dV$$

$$\partial_{ik} \hat{u}_k = -\frac{3\eta}{2a} \hat{u}_i; \quad \hat{\vec{z}}_0 = \frac{3a}{4} \frac{[\hat{u} \times \vec{z}]}{z^3}$$

$$\int \partial_{ik} h_k dS = 0$$

$$\vec{u} = \frac{1}{8\pi\eta} \int \frac{\vec{z} \times \vec{u}}{z^3} dV$$

Boundary layer approximation

$$u = \frac{\tau_f}{8\pi\eta a^2} K_u$$

τ_f - torque of rotary motor

$$\vec{e}_\omega = \vec{u}/u; \quad \vec{u} = u \vec{e}_u$$

$$K_u = \frac{1}{4\pi} \int \vec{e}_u \times \vec{e}_\omega \cdot \vec{e}_z d\omega$$

Equivalent approach

$$\vec{\tau}_z = \frac{3}{2} n \tilde{c}_f K_u \sin \nu \vec{e}_\varphi$$

Boundary layer

$$\frac{\partial \Omega_z}{\partial z} - \frac{1}{2} \frac{\partial \tilde{c}_\varphi}{\partial z} = 0 \Rightarrow \Omega_{\text{tot}} = \frac{1}{2} \tilde{c}_\varphi \text{ (zero total force)}$$

$$u_s(r) - u_s(z) = \frac{1}{2\eta} \int_0^z \tilde{c}_\varphi dz'$$

Slip velocity

$$|u_s| = \frac{3N \tilde{c}_f K_u \sin \nu}{4\eta} = u_0 \sin \nu \text{ (squeeze mode)}$$

Solution

$$u = \frac{2}{3} u_0 = \frac{N \tilde{c}_f K_u}{8\eta a^2}$$

Angular velocity

$$\Omega_i = \frac{1}{16\pi\eta} \int \left(\frac{3z_i (\vec{r} \cdot \vec{z})}{25} - \frac{\tilde{c}_i}{z^3} \right) dV$$

$$\vec{e}_{\Omega_i} \cdot \vec{\Omega} = \frac{N \tilde{c}_f}{16\pi\eta a^3} K_{\Omega}$$

Fig 2

$$K_{\Omega} = \frac{1}{4\pi} \int (3 (\vec{e}_z \cdot \vec{e}_r) (\vec{e}_{\Omega} \cdot \vec{e}_z) - \vec{e}_{\Omega} \cdot \vec{e}_r) d\omega$$

Estimates

$$\tilde{c}_f = 5 \text{ pN} \cdot \mu\text{m}; \quad a = 8.5 \mu\text{m}; \quad \eta = 1 \text{ cP}; \quad N = 200$$

$$u = \frac{N \tilde{c}_f}{8\eta a^2} \approx 550 \mu\text{m/s}$$

$$\Omega = \frac{N \tilde{c}_f}{16\pi\eta a^3} \approx 32 \text{ rad/s}$$

Rotating Crystals -7-

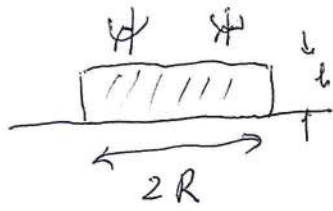


Fig. 4



$$\vec{\omega} = \Omega_0 \vec{e}_z \theta(R-z) \theta(h-z)$$

$$\Omega_0 < 0$$

$$v_\varphi = \frac{\hbar \alpha \Omega_0 R}{2\pi} \int_0^\infty \frac{J_1(kR) J_1(kz)}{k} \left(1 - e^{-kz} - e^{-kh} \operatorname{sh}(kz) \right) dk$$

$$\Omega = v_\varphi|_{z=R}/R; \quad \pi = 2\pi/\Omega$$